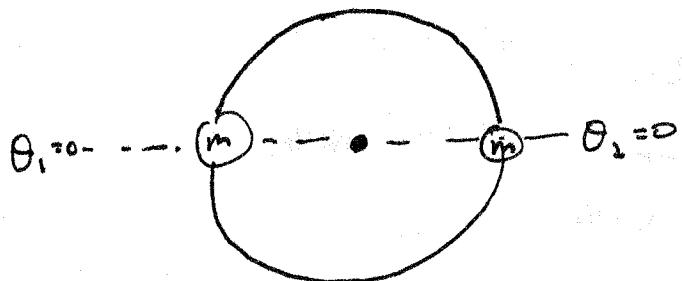


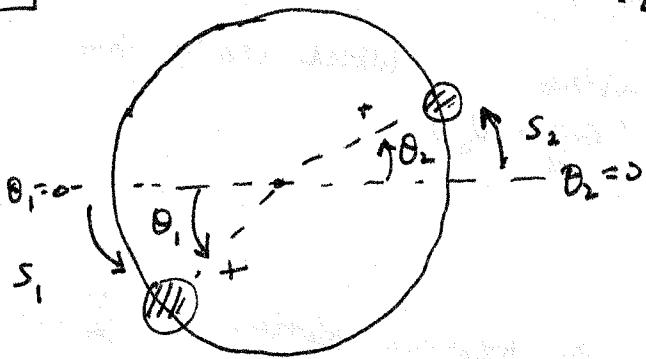
Problem 1 Ring Oscillator

I(a) In equilibrium:  $\theta_1 = 0$ ,  $\theta_2 = 0$



No net force on either bead.

I(b)



Let  $s$  be arc length

$$s_1(t) = R\theta_1(t)$$

$$s_2(t) = R\theta_2(t)$$

(We're measuring the arc length between a bead at its present position  $\theta(t)$  and its equilibrium position  $\theta=0$ .)

Then Newton's 2nd law, applied to motion along the circle becomes:

~~Newton's 2nd law~~  $m\ddot{s}_1 = -k(s_1 - s_2) + k(s_2 - s_1) \dots \underline{\text{Eqn 1}}$

$m\ddot{s}_2 = k(s_1 - s_2) - k(s_2 - s_1) \dots \underline{\text{Eqn 2}}$

But  $s_j = R\theta_j$  so:

$mR\ddot{\theta}_1 = -kR(\theta_1 - \theta_2) + kR(\theta_2 - \theta_1) \dots \underline{\text{Eqn 1}}$

$mR\ddot{\theta}_2 = +kR(\theta_1 - \theta_2) - kR(\theta_2 - \theta_1) \dots \underline{\text{Eqn 2}}$

Rearranging:

$$\left. \begin{aligned} \ddot{\theta}_1 &= -2\omega_0^2\theta_1 + 2\omega_0^2\theta_2 \\ \ddot{\theta}_2 &= 2\omega_0^2\theta_1 - 2\omega_0^2\theta_2 \end{aligned} \right\} \dots \underline{\text{Eqn 1}} \quad \underline{\text{Eqn 2}}$$

$$\omega_0 = \sqrt{\frac{k}{m}}$$

(d)

(C) From (b), the EOMs are:

Pg 2

$$\ddot{\theta}_1 + 2\omega_0^2 \theta_1 - 2\omega_0^2 \theta_2 = 0 \quad \dots \quad (1)$$

$$\ddot{\theta}_2 - 2\omega_0^2 \theta_1 + 2\omega_0^2 \theta_2 = 0 \quad \dots \quad (2)$$

1: Adding EOMs (1) & (2) together:

$$(\ddot{\theta}_1 + \ddot{\theta}_2) = 0. \quad \text{Let} \quad \dot{\Theta} \equiv \dot{\theta}_1 + \dot{\theta}_2.$$

$$\Rightarrow \boxed{\ddot{\Theta} = 0}$$
 is "Effective EOM" above equation.  $\dot{\Theta}$  "Big theta"

Solution of this EOM is found by noticing that the acceleration  $\ddot{\Theta}$  is zero.  $\Rightarrow$  constant velocity motion (Call it  $V_0$ )

Some initial position  $\Theta_0$  at  $t=0$ .

$$\Rightarrow \boxed{\Theta(t) = V_0 t + \Theta_0} \quad \leftarrow \text{General solution to } \ddot{\Theta} = 0$$

$(V_0, \Theta_0$  : free parameters)

You can check that this is indeed solution to  $\ddot{\Theta} = 0$  by plugging it back into the eqn  $\ddot{\Theta} = 0$ .

It's also the general solution to  $\dot{\Theta} = 0$

Since it has 2 free parameters  $V_0$  and  $\Theta_0$ .  $\ddot{\Theta} = 0$  is a 2nd order differential eqn.

Next,

2: Subtracting EOMs ① & ② on Pg 2:

(Pg 3)

$$(\ddot{\theta}_1 - \ddot{\theta}_2) + 4\omega_0^2 \theta_1 - 4\omega_0^2 \theta_2 = 0$$

$$\Rightarrow \boxed{(\ddot{\theta}_1 - \ddot{\theta}_2) + 4\omega_0^2 (\theta_1 - \theta_2) = 0}$$

Let:

$$\Omega \equiv \theta_1 - \theta_2.$$

then:

2 "Big omega"

$$\rightarrow \boxed{\ddot{\Omega} + 4\omega_0^2 \Omega = 0}$$

↑ "Effective EOM" 2

But this is just the EOM of a simple harmonic oscillator with angular frequency  $\tilde{\omega}$ :

- we can thus immediately write down the general solution to above EOM;

$$\tilde{\omega} = \sqrt{4\omega_0^2}$$

$$= 2\omega_0$$

$$\boxed{\Omega(t) = C \cos(2\omega_0 t - \phi).}$$

← General solution to the effective EOM 1/2.

(C,  $\phi$ : free parameters)

Now, using the solutions to the 2 effective EOMs, we can obtain general solutions describing  $\theta_1(t)$  and  $\theta_2(t)$ :

since  $\Omega = \theta_1 - \theta_2$

$$\Theta = \theta_1 + \theta_2$$

we have:

$$\begin{cases} \theta_1(t) = \frac{\Omega(t) + \Theta(t)}{2} \\ \theta_2(t) = -\frac{\Omega(t) + \Theta(t)}{2} \end{cases}$$

thus,

$$\left. \begin{aligned} \theta_1(t) &= \frac{C}{2} \cos(2\omega_0 t - \phi) + \frac{V_0 t}{2} + \frac{\tilde{\Theta}_0}{2} \\ \theta_2(t) &= -\frac{C}{2} \cos(2\omega_0 t - \phi) + \frac{V_0 t}{2} + \frac{\tilde{\Theta}_0}{2} \end{aligned} \right\}$$

But  ~~$\frac{1}{2}$~~  to avoid keep writing  $\frac{1}{2}$ , we define

$$\tilde{C} = \frac{1}{2}, \quad \tilde{V}_0 = V_0/2, \quad \tilde{\Theta}_0 = \Theta_0/2$$

And writing in vector form:

$$\Rightarrow \begin{pmatrix} \theta_1(t) \\ \theta_2(t) \end{pmatrix} = (\tilde{V}_0 t + \tilde{\Theta}_0) \begin{pmatrix} 1 \\ 1 \end{pmatrix} + \tilde{C} \begin{pmatrix} 1 \\ -1 \end{pmatrix} \cos(2\omega_0 t - \phi)$$

$\tilde{V}_0, \tilde{\Theta}_0, \tilde{C}, \phi$  are 4 free parameters.

There are 2 normal modes: ~~for each particle~~ ~~for each mode added together, eg. 2~~

Normal mode #1:  $\omega_1 = 0 \leftarrow$  Normal mode angular frequency!

$$(\tilde{V}_0 t + \tilde{\Theta}_0) \begin{pmatrix} 1 \\ 1 \end{pmatrix} \equiv V_1$$

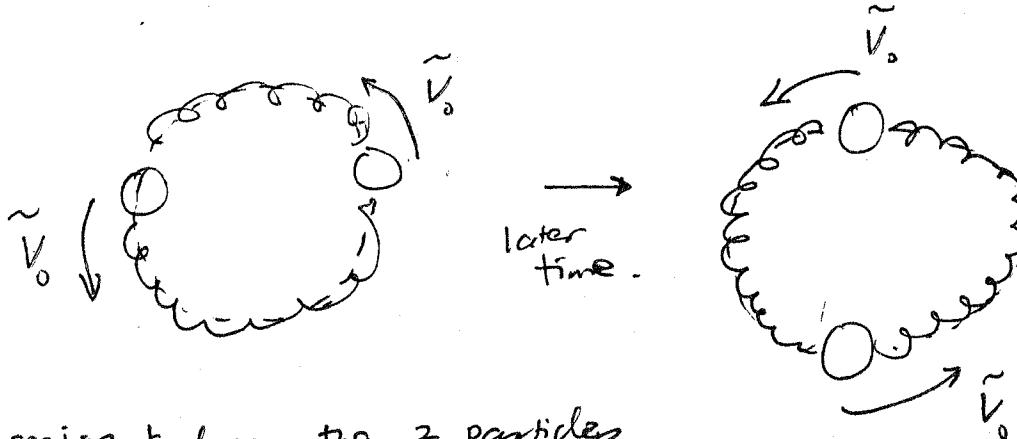
$\uparrow$  "defined as"

2 free parameters  
for each particle  
if suitable  
M of  
parameters

Normal mode #2:  $\omega_2 = 2\omega_0 \leftarrow$  Normal mode angular frequency:

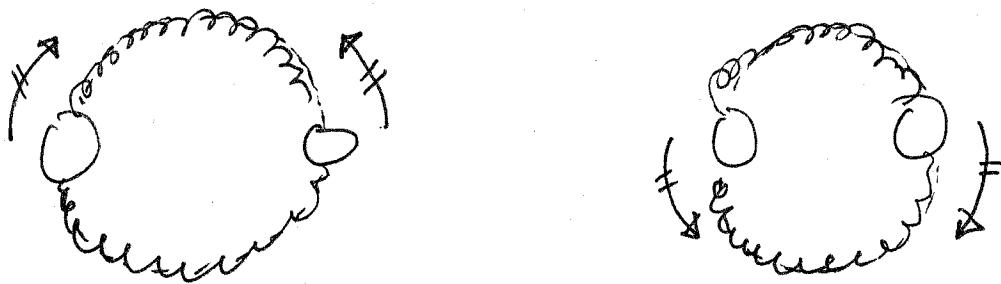
$$V_2 = \tilde{C} \begin{pmatrix} 1 \\ -1 \end{pmatrix} \cos(2\omega_0 t - \phi)$$

- Normal mode #1 describes pure rotation of the 2 particles in a circle, with angular ~~speed~~ velocity  $\tilde{\nu}_0$ :



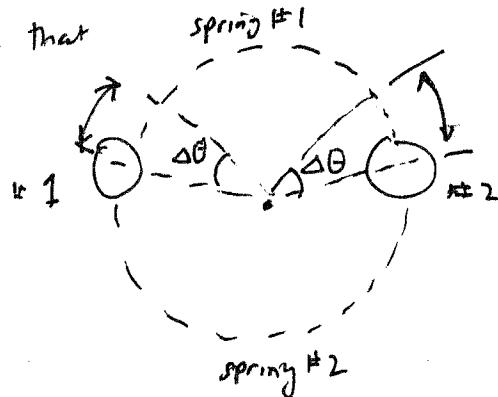
The springs between the 2 particles remain Unstretched / compressed relative to equilibrium configuration at all times.

- Normal mode #2 describes anti-symmetric oscillation of 2 particles.



2 particles move towards each other w/ same angular speed.

Notice that spring #1



when #1 moves by  $\Delta\theta$ , so does #2.

$\Rightarrow$  Spring #1 compressed by  $2(\Delta\theta)R$ .

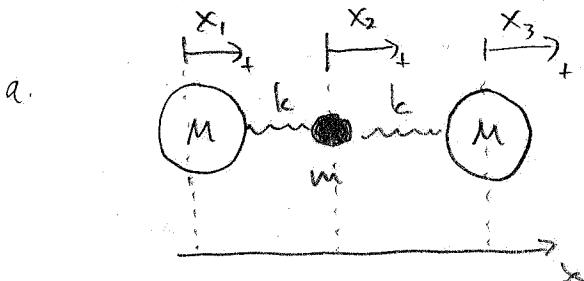
Also, spring #2 is stretched by  $2(\Delta\theta)R$ . at the same time.

$\Rightarrow$  so this is equivalent to the following one particle system:

$$\Rightarrow \boxed{\omega_2 = \sqrt{\frac{4k}{m}} = 2\sqrt{\frac{15}{m}} = 2\omega_0}$$



## Problem 2 Solution

 $x_n$  = displacement from equilibrium

@  $x_1 = 0 \Rightarrow$  corresponding mass is at equilibrium (no net forces).

Whenever @ equilibrium, we define the displacement as zero.

b. Atom 1:  $\sum F_1 = M\ddot{x}_1 = k(x_2 - x_1)$

if  $x_2 - x_1 > 0 \Rightarrow$  spring is stretched  
 $\Rightarrow$  restoring force in + direction

Atom 2:  $\sum F_2 = m\ddot{x}_2 = -k(x_2 - x_1) + k(x_3 - x_2)$

Atom 3:  $\sum F_3 = M\ddot{x}_3 = -k(x_3 - x_2)$

if  $x_3 - x_2 > 0 \Rightarrow$  spring is stretched

$\Rightarrow$  restoring force in (-) direction

c. EOM<sub>1</sub>:  $\ddot{x}_1 + \frac{k}{M}(x_1 - x_2) = 0$

EOM<sub>2</sub>:  $\ddot{x}_2 + \frac{k}{m}(-x_1 + 2x_2 - x_3) = 0$

EOM<sub>3</sub>:  $\ddot{x}_3 + \frac{k}{M}(-x_2 + x_3) = 0$

$$\left\{ \begin{array}{l} w_1^2 \equiv \sqrt{\frac{k}{M}} \\ w_2^2 \equiv \sqrt{\frac{k}{m}} \end{array} \right.$$

a.  $-w_1^2 x_1 + w_1^2 x_2 + 0 \cdot x_3 = \ddot{x}_1$

$w_2^2 x_1 + 2w_2^2 x_2 + w_2^2 x_3 = \ddot{x}_2$

$0 \cdot x_1 + w_1^2 x_2 - w_1^2 x_3 = \ddot{x}_3$

} from equations to matrix form

$$\begin{pmatrix} -w_1^2 & w_1^2 & 0 \\ w_2^2 & -2w_2^2 & w_2^2 \\ 0 & w_1^2 & -w_1^2 \end{pmatrix} \begin{pmatrix} x_1 \\ x_2 \\ x_3 \end{pmatrix} = \begin{pmatrix} \ddot{x}_1 \\ \ddot{x}_2 \\ \ddot{x}_3 \end{pmatrix}$$

(1)  
matrix P

d. Guess:  $\vec{z}(t) = \begin{pmatrix} A \\ B \\ C \end{pmatrix} e^{i\omega t} \Rightarrow \ddot{\vec{z}} = -\omega^2 \begin{pmatrix} A \\ B \\ C \end{pmatrix} e^{i\omega t}$

$$\Rightarrow P \cdot \underbrace{\begin{pmatrix} A \\ B \\ C \end{pmatrix} e^{i\omega t}}_{\vec{z}(t)} = \underbrace{-\omega^2 \begin{pmatrix} A \\ B \\ C \end{pmatrix} e^{i\omega t}}_{\ddot{\vec{z}}(t)} \quad \text{Plug into complex EOM}$$

$$\Rightarrow P \cdot \begin{pmatrix} A \\ B \\ C \end{pmatrix} = - \begin{pmatrix} \omega^2 & 0 & 0 \\ 0 & \omega^2 & 0 \\ 0 & 0 & \omega^2 \end{pmatrix} \begin{pmatrix} A \\ B \\ C \end{pmatrix} \quad \text{used identity matrix } \begin{pmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{pmatrix}$$

$$\Rightarrow P \cdot \begin{pmatrix} A \\ B \\ C \end{pmatrix} + \begin{pmatrix} \omega^2 & 0 & 0 \\ 0 & \omega^2 & 0 \\ 0 & 0 & \omega^2 \end{pmatrix} \begin{pmatrix} A \\ B \\ C \end{pmatrix} = \vec{0} \quad ; \quad \vec{0} = \begin{pmatrix} 0 \\ 0 \\ 0 \end{pmatrix}$$

$$\Rightarrow \left[ \begin{pmatrix} -\omega_1^2 & \omega_1^2 & 0 \\ \omega_2^2 & -2\omega_2^2 & \omega_2^2 \\ 0 & \omega_1^2 & -\omega_1^2 \end{pmatrix} + \begin{pmatrix} \omega^2 & 0 & 0 \\ 0 & \omega^2 & 0 \\ 0 & 0 & \omega^2 \end{pmatrix} \right] \begin{pmatrix} A \\ B \\ C \end{pmatrix} = \vec{0}$$

$$\Rightarrow \underbrace{\begin{pmatrix} -\omega_1^2 + \omega^2 & \omega_1^2 & 0 \\ \omega_2^2 & -2\omega_2^2 + \omega^2 & \omega_2^2 \\ 0 & \omega_1^2 & -\omega_1^2 + \omega^2 \end{pmatrix}}_{P_1} \begin{pmatrix} A \\ B \\ C \end{pmatrix} = \vec{0}$$

where  $\det(P_1) = 0$ , we can find non-trivial solns b/c it means  $P_1$  has no inverse  $P_1^{-1}$ .  
 Remember, trivial soln is when  $A=B=C=0$ , and this is not very useful.

$$\det(P_1) = (-\omega_1^2 + \omega^2) \begin{vmatrix} -2\omega_2^2 + \omega^2 & \omega_2^2 \\ \omega_1^2 & -\omega_1^2 + \omega^2 \end{vmatrix} - \omega_1^2 \begin{vmatrix} \omega_2^2 & \omega_2^2 \\ 0 & -\omega_1^2 + \omega^2 \end{vmatrix} = 0$$

If  $\det(P_1) = 0 \Rightarrow$  matrix cannot be inverted ( $P_1^{-1}$  does not exist).

If  $P_1$  existed, then  $P_1^{-1}P_1 \begin{pmatrix} A \\ B \\ C \end{pmatrix} = P_1^{-1} \begin{pmatrix} 0 \\ 0 \\ 0 \end{pmatrix} = 0 \Rightarrow \begin{pmatrix} A \\ B \\ C \end{pmatrix} = \begin{pmatrix} 0 \\ 0 \\ 0 \end{pmatrix}$  (not useful)

$$0 = \det(P_1) = (-w_1^2 + \alpha^2) \left[ (-2w_2^2 + \alpha^2)(-w_1^2 + \alpha^2) - w_1^2 w_2^2 \right] - w_1^2 [w_2^2(-w_1^2 + \alpha^2) - 0]$$

$$= (-w_1^2 + \alpha^2) \left[ w_1^2 w_2^2 - (2w_2^2 + w_1^2)\alpha^2 + \alpha^4 \right] - w_1^2 [-w_1^2 w_2^2 + w_2^2 \alpha^2] = 0$$

$$0 = -w_1^4 w_2^2 + (2w_1^2 w_2^2 + w_1^4) \alpha^2 - w_1^2 \alpha^4 + w_1^2 w_2^2 \alpha^2 - (2w_2^2 + w_1^2) \alpha^4 + \alpha^6 + w_1^4 w_2^2 - w_1^2 w_2^2 \alpha^2$$

$$\alpha^6 - (2w_2^2 + 2w_1^2) \alpha^4 + (2w_1^2 w_2^2 + w_1^4) \alpha^2 = 0 \quad \leftarrow \text{Eqn ①}$$

$$\Rightarrow \alpha^4 - (2w_2^2 + 2w_1^2) \alpha^2 + (2w_1^2 w_2^2 + w_1^4) = 0 \quad ; \quad \text{let some } x = \alpha^2$$

$$\Rightarrow x^2 - 2(w_1^2 + w_2^2)x + (2w_1^2 w_2^2 + w_1^4) = 0$$

$\leftarrow$  we can use the quadratic equation to find  $x$

$$\frac{2(w_1^2 + w_2^2) \pm \sqrt{4(w_1^4 + 2w_1^2 w_2^2 + w_2^4) - 4(2w_1^2 w_2^2 + w_1^4)}}{2}$$

$$= 2w_1^2 + w_2^2 \pm 2\sqrt{w_1^4 - w_1^4 + 2w_1^2 w_2^2 - 2w_1^2 w_2^2 + w_2^4}$$

$$\alpha^2 = w_1^2 + w_2^2 \pm w_2^2 = x$$

$$\Rightarrow \alpha^2 = w_1^2 \quad \text{or} \quad \alpha^2 = w_1^2 + 2w_2^2$$

$$\Rightarrow \boxed{\alpha = \pm w_1} \quad \text{or} \quad \boxed{\alpha = \pm \sqrt{w_1^2 + 2w_2^2}} \quad \left\{ \begin{array}{l} 4 \text{ solutions, but Eqn ①} \\ \text{suggests there} \\ \text{should be 6,} \end{array} \right.$$

well  $\alpha^2 = 0$  is also a solution

$$\Rightarrow \boxed{\alpha = 0} \quad \text{or} \quad \boxed{\alpha = 0}$$

2 more solutions

$\Rightarrow$  There are 5 possible values for  $\alpha$  (six if you take redundancy into account), which makes sense given  $\alpha$  rises to the order of 6 ( $\alpha^6$ ) in Eqn ①

e.  $\alpha = 0$  does not correspond to a vibrational mode.

In our guess for  $\mathbf{z}(t)$ ,  $e^{i\omega t} = 1 \Rightarrow$  no oscillation

$\Rightarrow \alpha = 0$  corresponds to a translational normal mode where all three atoms displace in the same direction with equal constant velocity.

$$V_5 := \begin{pmatrix} A \\ B \\ C \end{pmatrix} e^{i\omega t} = \begin{pmatrix} A \\ B \\ C \end{pmatrix}, \text{ in (f.) we find that } A = B = C$$

$$\Rightarrow V_5 = A_5 \begin{pmatrix} 1 \\ 1 \\ 1 \end{pmatrix} \quad [A_5] = \text{length}$$

We could also guess  $\mathbf{z}(t) = t \begin{pmatrix} A \\ B \\ C \end{pmatrix}$  to be a solution to the EOM, and we'd find it to be true, and you'd also find that  $A = B = C$  (you can try it to prove it to yourself...)

$$\Rightarrow V_6 = B_6 \begin{pmatrix} 1 \\ 1 \\ 1 \end{pmatrix} t \quad [B_6] = \frac{\text{length}}{\text{time}}$$

$\Rightarrow$  the normal mode associated with  $\alpha = 0$  can be written as follows:

(we will call this  $\alpha_3$ ,

$$\alpha_3 = 0$$

$$V_5 + V_6 = N_{\alpha_3}$$

f.  $\omega = \pm \omega_1$  ← we examine  $\omega = \pm \omega_1$  b/c  $\omega^2 = \omega_1^2$  for either value of  $\omega$

$$P_1 = \begin{pmatrix} 0 & \omega_1^2 & 0 & 0 \\ \omega_2^2 & -2\omega_2^2 + \omega_1^2 & \omega_2^2 & 0 \\ 0 & \omega_1^2 & 0 & 0 \end{pmatrix} \Rightarrow \begin{pmatrix} 0 & \omega_1^2 & 0 & 0 \\ \omega_2^2 & -2\omega_2^2 + \omega_1^2 & \omega_2^2 & 0 \\ 0 & \omega_1^2 & 0 & 0 \end{pmatrix} \begin{pmatrix} A \\ B \\ C \end{pmatrix} = \vec{0}$$

$$\Rightarrow B\omega_1^2 = 0 ; A\omega_2^2 + (-2\omega_2^2 + \omega_1^2)B + C\omega_2^2 = 0 ; B\omega_1^2 = 0$$

if

$$B=0 \Rightarrow A\omega_2^2 + C\omega_2^2 = 0 \Rightarrow A=-C, C=-A$$

$$\text{so, } \boxed{B=0, C=-A}$$

$$\underline{\omega = \pm \sqrt{\omega_1^2 + 2\omega_2^2}}$$

$$P_1 = \begin{pmatrix} 2\omega_2^2 & \omega_1^2 & 0 \\ \omega_2^2 & \omega_1^2 & \omega_2^2 \\ 0 & \omega_1^2 & 2\omega_2^2 \end{pmatrix} \Rightarrow \begin{pmatrix} 2\omega_2^2 & \omega_1^2 & 0 \\ \omega_2^2 & \omega_1^2 & \omega_2^2 \\ 0 & \omega_1^2 & 2\omega_2^2 \end{pmatrix} \begin{pmatrix} A \\ B \\ C \end{pmatrix} = \vec{0}$$

$$\Rightarrow 2\omega_2^2 A + \omega_1^2 B = 0 ; A\omega_2^2 + B\omega_1^2 + C\omega_2^2 = 0 ; B\omega_1^2 + 2\omega_2^2 C = 0$$

$$B = -\frac{2\omega_2^2 A}{\omega_1^2} \Rightarrow -A\omega_2^2 + C\omega_2^2 = 0$$

$$\Rightarrow C = A$$

substitute for  $\omega_1^2, \omega_2^2$

$$B = \pm \frac{2M}{m} A ; M > m \Rightarrow |B| > |A|$$

$$\Rightarrow \boxed{B = -\frac{2\omega_2^2}{\omega_1^2} A = -\frac{2M}{m} A, C = A}$$

f.  $\alpha = 0$

$$P_1 = \begin{pmatrix} -\omega_1^2 & \omega_1^2 & 0 \\ \omega_2^2 & -2\omega_2^2 & \omega_2^2 \\ 0 & \omega_1^2 & -\omega_1^2 \end{pmatrix} \Rightarrow \begin{pmatrix} -\omega_1^2 & \omega_1^2 & 0 \\ \omega_2^2 & -2\omega_2^2 & \omega_2^2 \\ 0 & \omega_1^2 & -\omega_1^2 \end{pmatrix} \begin{pmatrix} A \\ B \\ C \end{pmatrix} = \begin{pmatrix} 0 \\ 0 \\ 0 \end{pmatrix}$$

$$\Rightarrow -Aw_1^2 + Bw_1^2 = 0 \Rightarrow \boxed{A = B}$$

$$\Rightarrow Bw_2^2 - Cw_2^2 = 0 \Rightarrow \boxed{B = C} \Rightarrow \boxed{A = B = C}$$

$$\Rightarrow Aw_1^2 - 2Bw_2^2 + Cw_2^2 = Aw_1^2 - 2Aw_2^2 + Aw_2^2 = 0$$

g. from part (f), we see we can write the following

for cases were  $\alpha = \omega_1$  +  $\alpha = -\omega_1$

$$V_1 = A_1 \begin{pmatrix} 1 \\ 0 \\ -1 \end{pmatrix} e^{i\omega_1 t} \quad \text{b/c } A = -C, B = 0$$

we can set  $A_1$ , thus it is a free parameter

$$V_2 = A_2 \begin{pmatrix} 1 \\ 0 \\ -1 \end{pmatrix} e^{-i\omega_1 t} \quad A_2 \text{ is also a free parameter}$$

$\Rightarrow$  normal mode associated with  $\alpha = \pm \omega_1$

call this  $\alpha_1, \alpha_1 = \pm \omega_1$

$$V_1 + V_2 = N_{\alpha_1} e^{\pm i\omega_1 t}$$

for  $\alpha = \pm \sqrt{\omega_1^2 + 2\omega_2^2}$

$$V_3 = A_3 \begin{pmatrix} 1 \\ -2\omega_2^2 \\ \omega_1^2 \end{pmatrix} e^{i(\sqrt{\omega_1^2 + 2\omega_2^2})t} \quad \text{b/c } A = C, B = \frac{-2\omega_2^2}{\omega_1^2} A$$

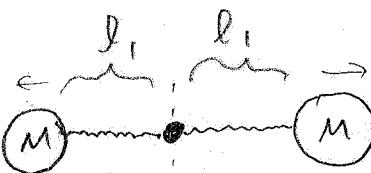
$$V_4 = A_4 \begin{pmatrix} 1 \\ -2\omega_2^2 \\ \omega_1^2 \end{pmatrix} e^{-i(\sqrt{\omega_1^2 + 2\omega_2^2})t}$$

$A_3 + A_4$  are free parameters

Now, let's draw what these look like...

$$\text{for } \alpha = \pm \omega_1$$

Snapshot 1:



Snapshot 2:



all oscillating with  
angular frequency  
of magnitude  $\omega_1$

Describes motion where the middle mass is stationary

& two outer masses oscillate in opposite directions with equal amplitude

$$e^{i\omega_1 t} = \underbrace{\cos(\omega_1 t)}_{\text{real part}} + i \underbrace{\sin(\omega_1 t)}_{\text{imaginary part}}$$

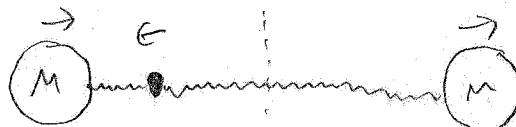
$$e^{-i\omega_1 t} = \underbrace{\cos(\omega_1 t)}_{\text{real part}} - i \underbrace{\sin(\omega_1 t)}_{\text{imaginary part}}$$

} the real parts of both solutions are equal

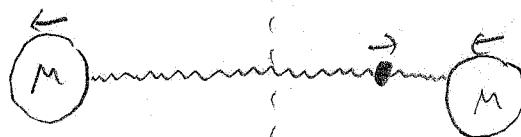
→ If we were to look at both cases  $\alpha = \omega_1$  or  $\alpha = -\omega_1$ , independently, we would not be able to see a difference in their motion

$$\text{for } \alpha = \pm \sqrt{\omega_1^2 + 2\omega_2^2}$$

Snapshot 1:



Snapshot 2:



all oscillating

with angular frequency  
 $\sqrt{\omega_1^2 + 2\omega_2^2}$

Describes a vibrational mode where the two outer masses oscillate in the same direction with equal amplitude. The middle mass oscillates in the opposite direction with increased amplitude by a factor of  $\frac{2M}{m}$ . All oscillate with angular frequency given by  $\alpha = \sqrt{\omega_1^2 + 2\omega_2^2}$

There will be no observable difference between cases where  $\alpha = \sqrt{\omega_1^2 + 2\omega_2^2}$   
or  $\alpha = -\sqrt{\omega_1^2 + 2\omega_2^2}$

for  $\lambda = 0$



Describes a translational mode where the three masses displace with equal constant velocity (in same direction)

$\Rightarrow$  normal mode associated with  $\omega = \pm \sqrt{\omega_1^2 + 2\omega_2^2}$

$$V_3 + V_4 = N_{\omega_2}$$

we will call this  
 $\omega_2, \omega_2 = \pm \sqrt{\omega_1^2 + 2\omega_2^2}$

Thus, we have 3 normal modes:  $N_{\omega_1}, N_{\omega_2}, N_{\omega_3}$

We expect to have 3 normal modes because we have 3 equations of motion.

Each equation of motion is a second-order linear differential equation, i.e. the highest derivative is the second derivative, thus we expect to have 2 free parameters for each of the 3 solutions to the 3 equations of motion  $\Rightarrow$  6 total free parameters

$N_{\omega_1} = V_1 + V_2$ ,  $V_1 + V_2$  each have a free parameter:  
 $A_1, A_2$ , respectively

$N_{\omega_2} = V_3 + V_4$ ,  $V_3 + V_4$  each have a free parameter:  
 $A_3, A_4$ , respectively

$N_{\omega_3} = V_5 + V_6$ ,  $V_5 + V_6$  each have a free parameter:  
 $A_5, A_6$ , respectively

$\Rightarrow$  general solution is the superposition of normal modes:

$$z_g(t) = N_{\omega_1} + N_{\omega_2} + N_{\omega_3} = V_1 + V_2 + V_3 + V_4 + V_5 + V_6$$



## Problem 3

## FROM MIDTERM SOLUTIONS

(3-1)

(a) The C-equivalent form is :  $\ddot{Z} + 2\gamma \dot{Z} + \omega_0^2 Z = \frac{F_0}{m} \cos(\omega t)$ ,

where  $Z(t)$  is a C-valued function.

(b) Before solving, represent:  $\cos(\omega t) = \frac{e^{i\omega t} + e^{-i\omega t}}{2}$ .

$$\Rightarrow \ddot{Z} + 2\gamma \dot{Z} + \omega_0^2 Z = \frac{F_0}{2m} \left\{ e^{i\omega t} + e^{-i\omega t} \right\}.$$

Then, solve:  $\ddot{Z}_1 + 2\gamma \dot{Z}_1 + \omega_0^2 Z_1 = \frac{F_0}{2m} e^{i\omega t}$

by guessing  $Z_1(t) = A e^{i\omega t}$  ← guess

Plugging in check:  $\ddot{Z}_1 + 2\gamma \dot{Z}_1 + \omega_0^2 Z_1$   
 $= -\omega^2 A e^{i\omega t} + 2\gamma i\omega e^{i\omega t} A + \omega_0^2 A e^{i\omega t}$

$$\Rightarrow \cancel{A e^{i\omega t}} [-\omega^2 + 2\gamma i\omega + \omega_0^2] = \frac{F_0}{2m} e^{i\omega t}$$

$$\Rightarrow \text{need: } A = \frac{F_0}{2m [-\omega^2 + 2\gamma i\omega + \omega_0^2]}$$

(Ans)



(C)

(3-2)

Linearity: If  $Z_1$  is sol'n to EOM, and so is  $Z_2$ , then  
 $Z_1 + Z_2$  is also a solution to EOM.

This is useful to ~~use~~ or since:

$Z_1(t)$  (found in (b)) is solution to:

$$\ddot{Z}_1 + 2\gamma \dot{Z}_1 + \omega_0^2 Z_1 = \frac{F_0}{2m} e^{i\omega t}$$

and if we have  $Z_2(t)$  being solution to: (we can get  $Z_2(t)$ )

$$\ddot{Z}_2 + 2\gamma \dot{Z}_2 + \omega_0^2 Z_2 = \frac{F_0}{2m} e^{-i\omega t} \quad \text{from } Z_1 \text{ by } j\sqrt{-1} \text{ changing } \omega \rightarrow -\omega$$

then

$Z_p = Z_1 + Z_2$  satisfies:

$$\ddot{Z}_p + 2\gamma \dot{Z}_p + Z_p \omega_0^2 = \frac{F_0}{2m} \left\{ e^{i\omega t} + e^{-i\omega t} \right\}$$

But  $Z_p(t)$  has no free parameters.

C-equivalent  
to our original eqn of motion

~~Note~~

But note that:

$$\left\{ \overset{\circ}{Z}_{\text{dohm}} + 2\gamma \overset{\circ}{Z}_{\text{dohm}} + \omega_0^2 Z_{\text{dohm}} = 0 \right\}$$

and

$$+ \left\{ \ddot{Z}_p + 2\gamma \dot{Z}_p + Z_p \omega_0^2 = \frac{F_0}{2m} \left\{ e^{i\omega t} + e^{-i\omega t} \right\} \right\}$$

$$\rightarrow \ddot{Z} + 2\gamma \dot{Z} + \omega_0^2 Z = \frac{F_0}{2m} \left\{ e^{i\omega t} + e^{-i\omega t} \right\}$$

where

$$Z = Z_{\text{dohm}} + Z_p$$

$$\leftarrow Z_{\text{dohm}}(t) = e^{-\gamma t} \left\{ A e^{i\tilde{\omega} t} + B e^{-i\tilde{\omega} t} \right\}$$

$\tilde{\omega}$  found in  $Q^{HL}$

So, using linearity again, we get the general soln :

(3-3)

$$Z(t) = e^{-\gamma t} \{ A e^{i\tilde{\omega}t} + B e^{-i\tilde{\omega}t} \}$$

$$+ \frac{F_0 e^{i\omega t}}{2m} \left\{ \frac{1}{-\omega^2 + 2i\gamma\omega + \omega_0^2} \right\}$$

$$+ \frac{F_0 e^{-i\omega t}}{2m} \left\{ \frac{1}{-\omega^2 - 2i\gamma\omega + \omega_0^2} \right\}$$

where  $\tilde{\omega} = \sqrt{\omega_0^2 - \gamma^2}$ . As D are free parameter.

(part f. on Pset #2)

$$X_p(t) = \frac{F_0}{m} \left\{ \frac{\omega^2 \cos(\omega t) + 2\gamma\omega \sin(\omega t)}{\omega^4 + 4\gamma^2\omega^2} \right\} \quad \dots \quad (1)$$

Want to write as  $X_p(t) = D(\omega) \cos(\tilde{\omega}t - \Theta)$ .

$$= D(\omega) \cos(\tilde{\omega}t) \cos(\Theta)$$

$$+ D(\omega) \sin(\tilde{\omega}t) \sin(\Theta) \quad \dots \quad (1)$$

So, matching  $\sin(\omega t)$  and  $\cos(\omega t)$  terms in eqn (1) with these :

We have :

$$D(\omega) \cos\Theta = \frac{F_0}{m} \frac{\omega^2}{\omega^4 + 4\gamma^2\omega^2} \quad \dots \quad (2)$$

$$D \sin\Theta = \frac{F_0}{m} \frac{2\gamma\omega}{\omega^4 + 4\gamma^2\omega^2} \quad \dots \quad (3)$$

$$\Rightarrow D^2 = \left( \frac{F_0}{m} \right)^2 \frac{\omega^4 + 4\gamma^2\omega^2}{(\omega^4 + 4\gamma^2\omega^2)^2} = \left( \frac{F_0}{m} \right)^2 \frac{1}{\omega^4 + 4\gamma^2\omega^2}$$

$$\Rightarrow D(\omega) = \frac{F_0}{m} \frac{1}{\sqrt{\omega^4 + 4\gamma^2\omega^2}}$$

over

To maximize  $D(\omega)$  : minimize the denominator

3-4

$$\Omega^4 + 4\gamma^2\omega^2.$$

$$\Rightarrow \frac{d}{d\omega} \left( \Omega^4 + 4\gamma^2\omega^2 \right)$$

$$\Omega \equiv \sqrt{\omega_0^2 - \omega^2}$$

$$= \frac{d}{d\omega} \left\{ (\omega_0^2 - \omega^2)^2 + 4\gamma^2\omega^2 \right\}$$

$$= 2(\omega_0^2 - \omega^2) (-2\omega) + 8\gamma^2\omega$$

$$\Rightarrow -4\gamma(\omega_0^2 - \omega^2) = -8\gamma^2\omega \quad (\omega \neq 0 \text{ since we have oscillating force})$$

$$\omega_0^2 - \omega^2 = 2\gamma^2$$

$$\Rightarrow \omega^2 = \omega_0^2 - 2\gamma^2$$

$$\Rightarrow \boxed{\omega_r = \sqrt{\omega_0^2 - 2\gamma^2}} \quad \xrightarrow{\text{Resonant angular frequency}}$$

(p)  $t > \frac{1}{\gamma}$  : Eqn (5) on midterm handout becomes:

$$X(t) \approx X_p(t).$$

$$\text{so: } \boxed{E_{\text{total}}(t) = \frac{1}{2} k X_p^{(t)}{}^2 + \frac{m \dot{X}_p^2}{2}}$$

13

(3-5)

f. (actually part d. on Pset #2)

We found  $z(t)$  to be as follows:

$$z(t) = e^{-\gamma t} \left[ A e^{i\hat{\omega}t} + B e^{-i\hat{\omega}t} \right] e^{\frac{F_0 e^{i\omega t}}{2m} \left[ \frac{1}{-\omega^2 + 2i\gamma\omega + \omega_0^2} \right]} + \frac{F_0 e^{-i\omega t}}{2m} \left[ \frac{1}{-\omega^2 - 2i\gamma\omega + \omega_0^2} \right]$$

we know the real part  
is  $(e^{-\gamma t} \cos(\omega t - \phi))$   
from our studies of under-  
damped harmonic oscillators  
 $\Rightarrow x_p(t) = (e^{-\gamma t} (\cos(\omega t - \phi)))$

Finding  $\operatorname{Re}[z_p(t)]$  will be a little tricky.

Finding  $\text{Re}[\epsilon_{\text{pp}}]$  because we have imaginary components (i) in

because we have imaginary numbers in the denominator. To remove i from the denominator, we will use a common strategy of multiplying the top & bottom of each term by what's called the complex conjugate...

A little background:  $n^*$  = complex conjugate of  $n$

$$i^* = -i \quad (\text{by definition})$$

if  $a$  is real,  $a^* = a$

$$\Rightarrow (-\omega^2 + 2i\gamma w + \omega_0^2)^* = -\omega^2 - 2i\gamma w + \omega_0^2 = \omega^2 - 2i\gamma w$$

$$\Rightarrow (-\omega^2 + 2i\gamma\omega + \omega_0^2)^* = -\omega^2 + 2i\gamma\omega + \omega_0^2 = \Omega^2 + 2i\gamma\omega$$

$$\Rightarrow \tilde{x}_p(t) = \frac{F_0 e^{i\omega t}}{2m} \left[ \frac{\Omega^2 - 2i\gamma\omega}{(\Omega^2 + 2i\gamma\omega)(\Omega^2 - 2i\gamma\omega)} \right]$$

$$+ \frac{F_0 e^{i\omega t}}{2m} \left[ \frac{\Omega^2 + 2i\gamma\omega}{(\Omega^2 - 2i\gamma\omega)(\Omega^2 + 2i\gamma\omega)} \right]$$

$$\Omega^2 = \omega_0^2 - \omega^2$$

← multiplied by  $\frac{\sqrt{z^2 - 2izw}}{\sqrt{z^2 - 2izw}} = 1$

$$\leftarrow \text{multiplied by } \begin{pmatrix} \bar{s^2 + 2is\omega} \\ \bar{s^2 + 2is\omega} \end{pmatrix} = \begin{pmatrix} 1 \\ 1 \end{pmatrix}$$

3-6

$$\Rightarrow z_p(t) = \frac{F_0 e^{i\omega t}}{2m} \left[ \frac{\Omega^2 - 2i\gamma\omega}{\Omega^4 + 4\gamma^2\omega^2} \right] + \frac{F_0 e^{-i\omega t}}{2m} \left[ \frac{\Omega^2 + 2i\gamma\omega}{\Omega^4 + 4\gamma^2\omega^2} \right]$$

$$\Rightarrow z_p(t) = \frac{F_0}{2m(\Omega^4 + 4\gamma^2\omega^2)} \left[ \Omega^2(e^{i\omega t} + e^{-i\omega t}) + 2i\gamma\omega(e^{-i\omega t} - e^{i\omega t}) \right]$$

common denominator (b/c two original denominators were complex conjugates of each other)

$$\Rightarrow z_p(t) = \frac{F_0}{2m(\Omega^4 + 4\gamma^2\omega^2)} \left[ \Omega^2(2\cos\omega t) + 2i\gamma\omega(2\cos\omega t - 2i\sin\omega t) \right]$$

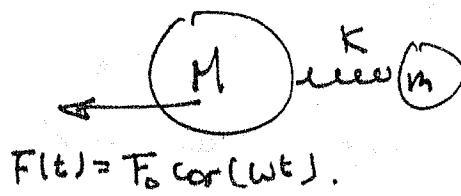
$$\Rightarrow x_p(t) = \frac{F_0}{2m(\Omega^4 + 4\gamma^2\omega^2)} \left[ \Omega^2(2\cos\omega t) + 4\gamma\omega\sin\omega t \right]$$

$$\Rightarrow x_p(t) = \frac{F_0 [\Omega^2 \cos\omega t + 2\gamma\omega \sin\omega t]}{2m [\Omega^4 + 4\gamma^2\omega^2]} \quad \checkmark \text{ this is real, so we can call it } x_p(t)$$

$\Rightarrow$  to find <sup>real</sup> general solution, add  $\checkmark x_h(t) + x_p(t)$

$$\Rightarrow \boxed{x(t) = (e^{-\gamma t} \cos(\tilde{\omega}t - \phi)) + \frac{F_0}{m} \frac{[\Omega^2 \cos\omega t + 2\gamma\omega \sin\omega t]}{[\Omega^4 + 4\gamma^2\omega^2]}}$$

$M \gg m$  ( $M$  is much larger mass than  $m$ )



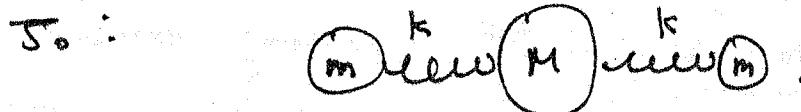
Let  
 $\omega_0 = \sqrt{\frac{k}{m}}$

$$F(t) = F_0 \cos(\omega t)$$

Suppose  $\omega = \omega_0$ . We want to show that if  $\omega = \omega_0$ , then the big particle (of mass  $M$ ) will be stationary.

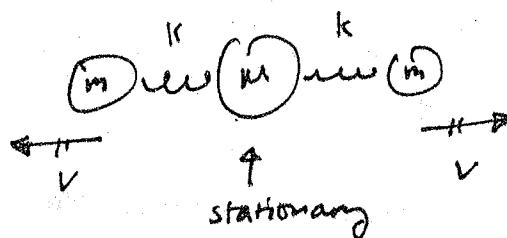
The key in this problem is to realize that we can have whatever physical mechanism providing the oscillatory force  $F(t)$ . (as long as  $F(t)$  oscillates with angular frequency  $\omega = \omega_0$ ).

One way to do this is by attaching another  $m$  to the left side of  $M$ , using ~~an~~ another spring of spring constant  $K$ .



To provide the oscillatory force, pull the left  $m$  by an amplitude  $A$ ; then release it. ~~the~~ the left  $m$  oscillates, it follows in the normal mode where we have the 2  $m$ 's oscillating antisymmetrically with respect to each other,

$M$  remain stationary.



The force that the left  $m$  exerts on  $M$  is:

$$F(t) = F_0 \cos(\omega_0 t)$$

$$= K A \cos(\omega_0 t)$$

since left  $m$  oscillates with frequency  $\omega_0$  in this normal mode, with amplitude  $A$ . Since  $A$  can be any amplitude we desire, we can exert any force on  $M$ .

$(K = \text{spring constant})$

(4-2)

From problem 2, you found that there are 3 normal modes.

One of them describes the normal mode with  $M$  stationary, described on previous page. We now need to show that  $M$  is stationary in the other 2 normal modes, then we'd be done w/ this problem. (since general motion of  $M$  is

the sum of the 3 normal modes, and thus if  $M$  doesn't move in all 3 normal modes, then the sum of them also describes  $M$  being stationary.)

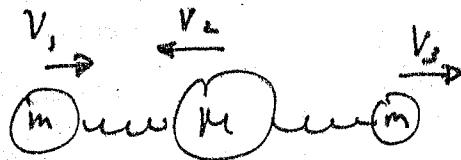
In another normal mode you found in problem 2:

$m \downarrow m \downarrow m \downarrow$

translational motion (No oscillations).

We discount this possibility by assuming that such translational motion is forbidden in this engineering system. (i.e. we're interested in knowing if  $M$  oscillates relative to ~~relative to~~ its right side  $m$ .)

Finally, the ~~only~~ other normal mode describes anti-symmetric oscillation of following type:



In your solution Problem 2, you should have found that if  $M \gg m$ , then  $v_2 \ll v_1$  and  $v_2 \ll v_3 \Rightarrow M$  is approximately stationary.

$\therefore$  we've shown that  $M$  remains stationary.

